

## FOUR-BAR MECHANISM FOR A PORTAL CRANE: A BRIEF OVERVIEW

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**Abstract:** The present article aims to increase knowledge of a four-bar mechanism. A four-bar system is a planar mechanism. There are many types of simple mechanisms, such as: Chebyshev, Chebyshev lambda, Hoecken, Roberts, Watt's, Grasshopper, horse-head, pantograph and Peaucelier. More complex mechanisms are: Stewart platform or Jansen's linkage. The determination of the degrees of freedom (DOF) for a system is done using the Chebychev-Gruebler-Kutzbach relationship. Due to this fact, the DOF for the four-bar system has the value one. Moreover, we analyze the following angles:  $\theta$ ,  $\alpha$ ,  $\beta$  and  $\varphi_3$  in the mechanism. Finally, using Matlab software the drawing and the dynamic simulation of a four-bar mechanism is carried out. Besides, the dynamic simulation is represented after 3s, 6s and 12s.

**Key words:** mechanism, linkage, DOF, angle, triangle.

### 1. INTRODUCTION

There are various mechanical mechanisms. These mechanisms can be: planar, spherical and spatial. In this paper, we studied just one system which is a planar mechanism.

The simplest mechanism known are: Chebyshev, Chebyshev lambda, Hoecken, Roberts, Watt's, Grasshopper, horse-head, pantograph and Peaucelier, Fig.1.

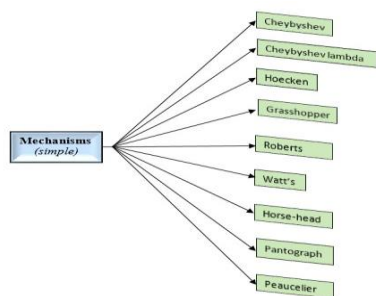


Figure 1 Simple mechanisms

Only Peaucelier and Grasshopper systems are exact straight-line mechanisms, [1].

There are also more complex mechanisms, such as: Stewart platform or Jansen's linkage.

In the field of mechanisms, a four-bar link is the simplest closed-chain movable link. It consists of four bodies, called bars or links. These bars are connected in a loop by four joints or pivots.

The four-bar mechanisms are widely used in various fields: medicine, research, education, transport, agriculture, industry, [2].

The movements of these systems is usually controlled and predictable, such as: rotation, oscillation and translation, Fig.2.

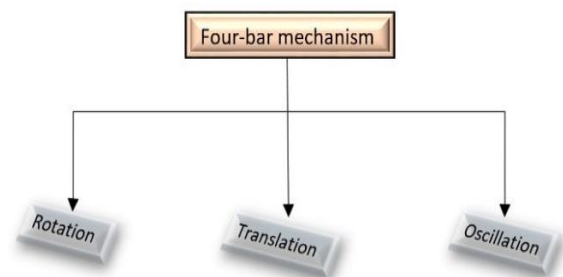


Figure 2 The movements of four-bar mechanism

### 2. STUDY ABOUT FOUR-BAR MECHANISM

In mechanical engineering, DOF is a number of independent motions that are allowed to the body. In the case of a mechanism made of several bodies, DOF represents the number of possible independent relative motions between the pieces of the system.

DOF is mostly studied in industrial robotics, [3] The Chebychev-Gruebler-Kutzbach formula determines the number for degrees of freedom:

$$DOF = 3(l - 1) - 2j \quad (1)$$

Where:

- DOF – degrees of freedom.
- $l$  – number of links.
- $j$  – number of joints

Then, DOF of the four-bar linkage can be calculated as follows:

$$DOF = 3(4 - 1) - 2 \cdot 4 = 3 \cdot 3 - 2 \cdot 4 = 9 - 8 = 1 \quad (2)$$

Where:

- $l =$  four links.
- $j =$  four joints.

The conditions of mechanisms and structures are:

- $DOF > 0$ , it is a mechanism.
- $DOF = 0$ , it is a structure.
- $DOF < 0$ , it is a preloaded structure.

From relation 2, DOF of the four-bar linkage is mechanism is 1, according to the conditions above,

A four-bar linkage is a mechanism having four rigid bars, Fig. 3:

- ❖ Frame is a fixed bar  $\rightarrow l_1$ .
- ❖ Crank (driver) is one rotate bar  $\rightarrow l_2$ .
- ❖ Connecting rod (coupler) is a floating link  $\rightarrow l_3$ .
- ❖ Rocker (follower) is other rotating bar  $\rightarrow l_4$ .

The connecting rod, however, does not rotate around a fixed center, [4].

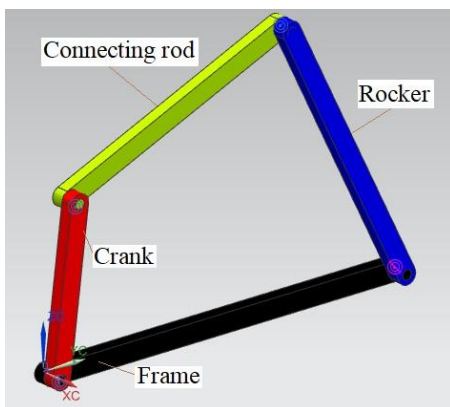


Figure 3 The linkages of mechanism

Furthermore, we studied the angles from a four-bar mechanism, Fig. 4.

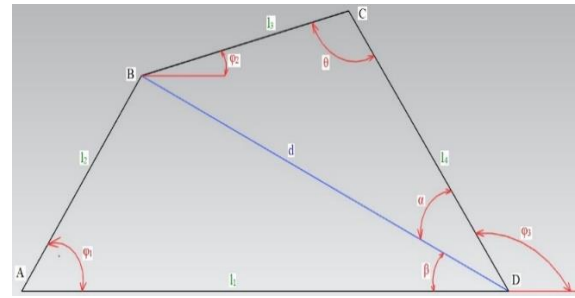


Figure 4 Angles from four-bar mechanism

By applying the law of cosine to triangle  $\Delta ABD$ , [5]:

$$d^2 = l_1^2 + l_2^2 - 2l_1l_2\cos\varphi_1 \quad (1)$$

Also, applying the law of cosine to triangle  $\Delta BCD$ :

$$d^2 = l_3^2 + l_4^2 - 2l_3l_4\cos\theta \quad (2)$$

Therefore,

$$l_1^2 + l_2^2 - 2l_1l_2\cos\varphi_1 = l_3^2 + l_4^2 - 2l_3l_4\cos\theta \quad (3)$$

And

$$\theta = \cos^{-1} \left( \frac{l_3^2 + l_4^2 - l_1^2 - l_2^2 + 2l_1l_2\cos\varphi_1}{2l_3l_4} \right) \quad (4)$$

Where:

- $\varphi_1$  – input angle.
- $\theta$  – transmission angle (this angle is only a function of the input angle  $\varphi_1$ ).

In the triangle  $\Delta BDC$ , the angle  $\alpha$  is determined with the relationship:

$$\alpha = \cos^{-1} \left( \frac{d^2 + l_4^2 - l_3^2}{2dl_4} \right) \quad (5)$$

In the triangle  $\Delta ABD$ , the angle  $\beta$  is determined with the relationship:

$$\beta = \cos^{-1} \left( \frac{d^2 + l_1^2 - l_2^2}{2dl_1} \right) \quad (6)$$

The last relationship is, [6]:

$$\varphi_3 = 180^\circ - (\alpha + \beta) \quad (7)$$

We used Matlab program to model a four-bar mechanism and then to simulate the dynamic behaviour of that system, Fig. 5.

```

16 % Calculations of angles and coordinates
17 for i = 1:length(theta)
18     G(i) = atan((a*sin(T(i)))/(d-a*cos(T(i))));
19     Phi(i) = acos((f(i)*f(1,i)+b*b-c*c)/(2*b*f(i))-G(i));
20     X(i) = a*cos(T(i))+e*cos(ar+Phi(i));
21     Y(i) = a*sin(T(i))+e*sin(ar+Phi(i));
22     hold on;
23     title('Four bar mechanism');
24     xlim([-50,110]);
25     ylim([-50,110]);
26     plot(X,Y,'b','LineWidth',1);
27
28     Cx = [0 a*cos(T(i)) b*cos(Phi(i))+a*cos(T(i)) d a*cos(T(i))+e*cos(Phi(i)+ar)];
29     Cy = [0 a*sin(T(i)) b*sin(Phi(i))+a*sin(T(i)) 0 a*sin(T(i))+e*sin(Phi(i)+ar)];
30
31     link_l1 = line([0 ,Cx(2)], [0 ,Cy(2)], 'LineWidth',4, 'Color', 'r');
32     link_l2 = line([Cx(2) ,Cx(3)], [Cy(2) ,Cy(3)], 'LineWidth',4, 'Color', 'g');
33     link_l3 = line([Cx(3) ,d], [Cy(3) ,0], 'LineWidth',4, 'Color', 'b');
34     link_l4 = line([0 ,d], [0 ,0], 'LineWidth',4, 'Color', 'k');
35     % Animation of mechanism
36     pause(0.08);
37     delete(link_l1);
38     delete(link_l2);
39     delete(link_l3);
40     delete(link_l4);
41 end

```

Figure 5 Calculations of angles and coordinates

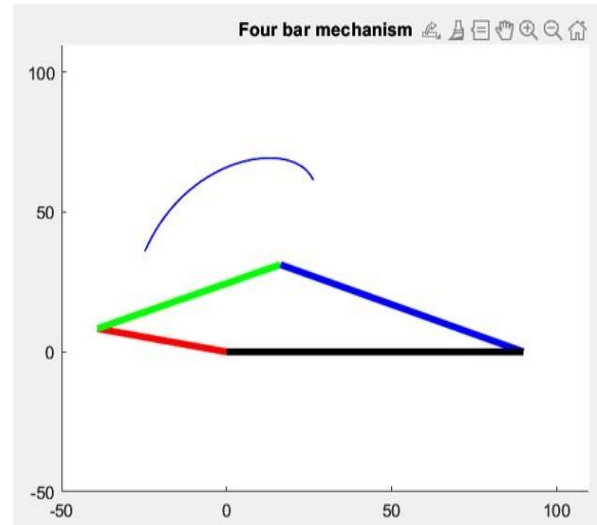


Figure 7 Four-bar mechanism after 6s

In this paper, the complete Matlab code built on top of the 41-line code, [7].

In the manuscript, a dynamic analysis was performed at three different time periods: 3s, 6s and 12s.

The simulation of a four-bar mechanism after three seconds is presented below, Fig. 6.

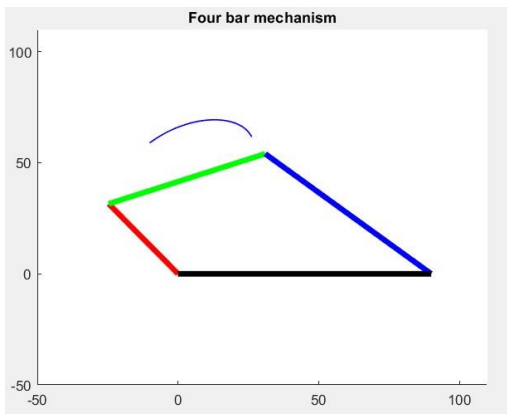


Figure 6 Four-bar mechanism after 3s

After that, the simulation of a four-bar mechanism after six seconds is presented below, Fig. 7.

In finally, the simulation of a four-bar mechanism after twelve seconds is presented below, Fig. 8.

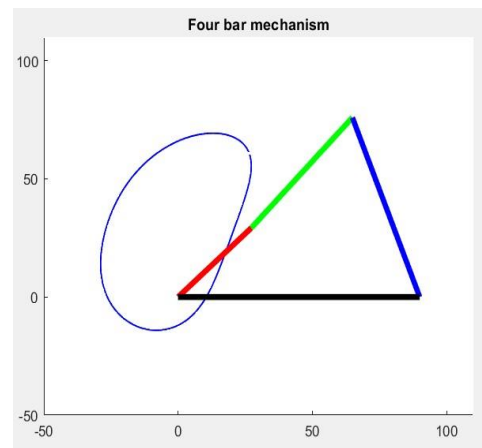


Figure 8 Four-bar mechanism after 12s

### 3. ACKNOWLEDGMENTS

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### 4. CONCLUSIONS

The four-bar mechanism occurs in many applications, such as in: robotics, automotive, pump jack, etc. Besides that, there are several advantages in using four-bar mechanisms:



- They are relatively simple, easy to design and manufacture, which can make them cost-effective and reliable.
- These systems have a large range of motion and can be used to create complex positions and movements.
- The systems can be used to create a variety of different configurations, such as crossed, hinged or parallel, which can be useful for different applications.
- The devices are relatively robust and can withstand high loads, making them suitable for use in a variety of applications.

10.1016/j.jest.2017.01.003, 4(3): 347-358.

However, in specialized universities, students must use various software (i.e. Nx Siemens, Solidworks, Autodesk Inventor, PTC Creo, etc.) to study four-bar.

In the future, we intend to develop a study of the four-bar mechanisms, especially constructed with linkages made of aluminum. This analysis will also include velocity and acceleration analysis.

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